

OPTIMIZATION OF STEEL PORTAL FRAMES USING PYTHON PROGRAMMING

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Abstract - The composition of any structure, be it is a bridge or building will be ample if it possible to construct with the needed economy and if throughout its useful life carries its intended loads and performs for intended function. Minimum weight design is design of frame which weight compatible mechanism can be formulated and a corresponding safe and statically admissible distribution of bending moment throughout the frame can be found. In this study an attempt is made to develop a software for minimum weight design of steel structures by using python programming. Plastic analysis is carried out for portal frames. Using the principle of kinematic theorem minimum weight design of portal fames carried out and graphical method of linear programming is used for optimization of weight. For testing the software performance, a single frame is considered and compared with manual calculations. It is clearly shows that there will be an accurate result in software compare to manual calculation and also it is a time constrained compare to manual calculation. Error percentage is also calculated and compared, it is having a thin difference in manual calculations.

Key Words: Optimisation, Plastic analysis, Linear programming, minimum weight, Portal frame

1. INTRODUCTION

The composition of any structure, be it is a bridge or building will be ample if it possible to construct with the needed economy and if throughout its useful life carries its intended loads and performs for intended function. Steel structures standout as more economical structure when compared with other sort of material structure. It has been realized that whenever members are rigidly connected, the structure has a much greater load carrying capacity than indicated by the elastic stress concept. Continuous or rigid frames are able to carry increased loads above first yield because structural steel has the capacity to yield in a ductile manner with no loss in strength. Thus, plastic design is a first design on the basis of the maximum load that structure can carry.

The idea of configuration of ultimate load design as the criteria is over 40 years of age. In the plastic analysis relative values that is minimum weight and minimum cost are not preassigned hence it is easy to adopt optimization. Optimization is the act of obtaining the best outcome under given conditions, optimization helps us to minimize the

weight of portal frames as well as cost, and in this work minimization of weight is focused.

1.1 INTRODUCTION TO PLASTIC ANALYSIS

Plastic analysis is based on the idealization of the stress strain curve as elastic perfectly plastic. The stress strain curve for a perfectly plastic material up to strain hardening is shown in the figure. Perfect plastic materials follow Hooks law up to the limit of proportionality. The stress-strain diagrams in compression and tension of the material are equal and also yield stresses in tension and compression are equal. The strains up to the strain hardening in tension and compression are equal. The stress strain curve show horizontal plateau both in tension and compression. Such materials are called perfectly plastic materials.



FIG.1.1. Stress-Strain curve for perfectly plastic materials

1.1.1 Assumptions for plastic moment sections

- The material obeys Hooke's law until the stress reaches the upper yield value on further straining, the stress drops to the lower yield value and thereafter remain constant.
- The yield stresses and the modulus of elasticity have the same value in compression as in tension.
- The material is homogeneous and isotropic in both the elastic and plastic states.
- The plane transverse sections remain plane and normal to the longitudinal axis after bending the effect of shear being neglected.
- Beam does not have resultant axial force.
- The cross section of the beam is symmetrical about an axis through its centroid parallel to plane of bending.
- Every layer of the material is free to expand and contract longitudinally and laterally under the stress as if separated from the other layers.

1.1.2 Fundamental conditions for plastic analysis:

- Mechanism condition: The ultimate or collapse load is reached when a mechanism is formed. The number of plastic hinges developed should be just sufficient to form a mechanism.
- Equilibrium condition: $\sum F_x = 0$, $\sum F_y = 0$, $\sum M_{xy} = 0$.
- Plastic moment condition: The bending moment at any section of the structure should not be more than the fully plastic moment of the section.

1.1.3 Theorems for plastic collapse:

- Lower bound theorem: A load factor computed on the basis of an arbitrary assumed bending moment diagram which is in equilibrium with the applied loads and where the fully plastic moment of resistance is nowhere exceeded will always be less than or at best equal to the load factor at rigid plastic collapse.
- Upper bound theorem: A load factor computed on the basis of an arbitrarily assumed mechanism will always be a greater than or at best equal to the load factor at rigid plastic collapse.
- Uniqueness theorem: If both the above criteria are satisfied then the resulting load factor corresponds to its value at rigid plastic collapse.

1.2 PLASTIC HINGES

Plastic hinges are formed when beam becomes fully plastic. At the plastic hinge an infinitely large rotation can occur under a constant moment equal to the plastic moment of the section. Plastic hinge is defined as a yielded zone due to bending in a structural member at which an infinite rotation can take place at a constant plastic moment M_p of the section. The number of hinges necessary for failure does not vary for a particular structure subject to a given loading condition although a part of a structure may fail independently by the formation of a smaller hinges. The member or structure behaves in the manner of a hinged mechanism and in doing so adjacent hinges rotate in opposite directions. Theoretically the plastic hinge is assumed to form at points at which plastic rotations occur.

1.3 OBJECTIVE FUNCTION.

The criteria with respect to which portal frames design is optimized, and it is expressed as a function of design variables is termed as Objective function. The choice of objective function is governed by nature of function. In the structural design the objective function is usually taken as minimization of cost or weight. Mathematically objective function is defined as

$$f(X) = \alpha_1 f_1(X) + \alpha_2 f_2(X)$$

Where α_1 and α_2 are constants and $f_1(X)$ and $f_2(X)$ are objective function.

2. METHODOLOGY

To establish the objectives of the study three things are major for minimum weight design

- Plastic analysis
- Minimization problem
- Linear programming

2.1 PLASTIC ANALYSIS

2.1.1 Methods of plastic analysis

Plastic designs are classified in to two types:

- a) Equilibrium method.
- b) Combining mechanisms method.

In this study combining mechanism method is performed to calculate the plastic analysis of portal frames. Combining mechanisms method is a type in which beam mechanism and sway mechanism are combined together to for a mechanism, this is shown in below figure 2.1



FIG.2.1 Portal frame mechanism

Indications of above figure

- a) Dimensions and loading. Units: m, kN. Plastic moments shown
- b) Beam mechanism
- c) Sway mechanism
- d) Combined mechanism

2.2 THE MINIMIZATION PROBLEM

Many feasible solutions can be determined for structural system and geometry. This fact is expressed by

mathematically in the existence of moreOunknowns. Furthermore, the distribution of moments over the structure at the ultimate load is influenced by the relative moment capacities of the various members. This fact becomes evident if the structure is observed during the last stages of loading leading to the ultimate load. As the ultimate load is approached, each succeeding hinge brings about a redistribution of moment which differs from the moment distribution that would have prevailed had the structure remained elastic.

At present this five-member frame, then, represents about the upper limit for trial procedures. The desired method should proceed from the first solution to the minimumweight solution with the consideration of only a very small percentage of the possible solutions.

2.3 LINEAR PROGRAMMING

The general linear programming problem can be stated in the following standard form:

1. Graphical method

The minimum weight design consists of one or two design variables, all the objective and constraint functions can be visualized with a 1-D or 2-D plot. One easily identify the optimal point from the graph.

The procedure of solving an optimization problem graphically as follows;

- a) Plot constraint boundaries and identify the feasible design region.
- b) Plot the contours of objective function and identify the direction along which the objective function increases or decreases.
- c) Identify the location of the optimal point. Usually the optimal point is located on one or more constraint boundaries. In other words, the optimal point is usually a tangent point between a contour of the objective function and some constraint boundaries.
- d) Solve the tangent point if needed.

3. MINIMUM WEIGHT DESIGN CONCEPT

3.1 Theorems for minimum weight design

3.1.1 Foulkes theorem:

"If for any design of a frame a weight compatible mechanism can be formulated, and a corresponding safe and statically admissible distribution of bending moment throughout the frame can be found, the design will be of minimum weight.

3.2 ASSUMPTIONS FOR MINIMUM WEIGHT DESIGN

- a) The frame is composed of uniform prismatic members.
- b) The infinite range of sections is available.
- c) Plastic collapse should occur under a single specified combination of factored loads.
- d) The linearized weight function is to be minimized.

3.3 EQUATIONS FOR MINIMUM WEIGHT DESIGN

Consider the minimum weight design of a fixed base rectangular portal frame We have 6 equations

- 1. $\beta_1 + \beta_2 = P^*L/2$
- 2. $\beta_1 = P^*L/4$
- 3. $\beta_2 = W^*a/4$
- 4. $\beta_1 + \beta_2 = W^* a/2$
- 5. $2\beta_2 + \beta_1 = (P^*L+W^*a)/2$
- 6. $2\beta_1 + \beta_2 = (P^*L + W^*a)/2$

Minimum weight solution:

$$X = \sum_{i=1}^{n} M p_{i}L_{i}$$

= M_{p1}L₁ + M_{p2}L₂ + M_{p3}L₃
= $\beta_{1}(2L) + 2\beta_{2}a$
X = $2\beta_{1}L + 2\beta_{2}a$

4. MINIMUM WEIGHT DESIGN SOFTWARE

4.1 User interface

The user interface of the software is shown below. The UI consists of Input values and objective function values for calculating minimum weight function.



FIG.4.1. User Interface

4.2 LINEAR PROGRAMMING GRAPH:

After entering the all input values we can generate the graph directly as shown in the figure below.

This graph consists of all the mechanisms equations which are already predefined in the script. From this graph we can easily get the feasible solutions and we can save the graph in many formats.



FIG.4.2. Linear programming graph of frame

4.3 MINIMUM WEIGHT SOLUTION

After plotting the LP graph we have to take all the points were lines intersect each other and calculate the objective solution "x" in the user interface box

4.4 FLOW CHART OF THE SOFTWARE



5. RESULTS AND DISCUSSION:

Computation of minimum weight solution is rendered manually as well as programmed software and results are compared. Two frames are considered for comparison each frame is having different dimensions and same assumed properties.

FRAME 1:

Data considered: Vertical Load (P): 10 KN Lateral load (W): 5 KN Span of beam: 5 m Span of column: 3m





Software calculation:

Step 1:

Input all the values



FIG.5.2. Input values

Step 2:

L

Generate graph and mark the all feasible solution points respectively.



Volume: 06 Issue: 02 | Feb 2019

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FIG.5.3. LP Graph

Feasible points are:

| X | Y |
|----|----|
| 4 | 25 |
| 18 | 8 |
| 13 | 13 |

Step 3:

Plot the all points in objective function table and get the values



FIG.5.4. Minimum weight solution

Minimum weight solution (x) = 132.5

Manual calculation:

| We ha | ve 6 equations |
|---------------|--|
| 1. | $\beta_1 + \beta_2 = P^*L/2$ |
| 2. | $\beta_1 = P^*L/4$ |
| 3. | $\beta_2 = W^*a/4$ |
| 4. | $\beta_1 + \beta_2 = W^*a/2$ |
| 5. | $2\beta_2 + \beta_1 = (P^*L + W^*a)/2$ |
| 6. | $2\beta_1 + \beta_2 = (P^*L + W^*a)/2$ |
| • | From equation 1 |
| β 1+ β | $_{2} = P^{*}L/2$ |
| Plotti | ng points are (25, 25) |
| • | From equation 2 |
| $\beta_1 = P$ | *L/4 |
| Plotti | ng points are (0, 12.5) |
| • | From equation 3 |
| $\beta_2 = V$ | V*a/4 |
| Plotti | ng points are (3.75, 0) |
| • | From equation 4 |

 $\beta_1 + \beta_2 = W^*a/2$ Plotting points are (7.5, 7.5) • From equation 5 $2\beta_2 + \beta_1 = (P^*L+W^*a)/2$ Plotting points are (16.25, 32.25) • From equation 6 $2\beta_1 + \beta_2 = (P^*L+W^*a)/2$ Plotting points are (32.25,16.25).

Plotting a graph as shown in below:



FIG.5.5. Manual graph

From the above graph: Feasible solution points are:

- 2) (7.2,18)
- 3) (12.5 ,12.5)

Minimum weight solution:

 $\begin{aligned} & X = \sum_{i=1}^{n} M p_{iL_i} \\ &= M_{p1}L_1 + M_{p2}L_2 \\ & \text{Minimum weight solution } (x) = 133.2 \end{aligned}$

5.2 CALCULATION OF ERROR PERCENTAGE:

| | Frame 1 |
|-------------|---------|
| Software | 132.5 |
| calculation | |
| Manual | 133.2 |
| calculation | |
| Error | 0.07% |
| percentage | |

6. CONCLUSIONS

1. An attempt is made to develop the software by using python programming to get the solution of minimum weight design. This software uses beam mechanism, sway mechanism and combined mechanism to

Volume: 06 Issue: 02 | Feb 2019

calculate all the values which require to plot in the linear programming graph.

- 2. As soon as graph is plotted we can easily get the feasible points in the software and then we can calculate the weight function respectively.
- 3. The software results are compared with manual calculation results by taking five portal frames with different dimensions.
- 4. The results which are getting from software calculation is more accurate than the manual calculation result. Linear programming graph can be plotted for any scales and any required values easily in software compared to manually plotting graph.
- 5. Error percentage is calculated; it is having teeth skin difference for portal frames having less weight as weight increasing manual calculation values have large errors compared to software calculation.
- 6. Software takes less time compared to manual calculation.

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