

EFFECT OF EXHAUST GAS RECIRCULATION ON PERFORMANCE AND EMISSION CHARACTERISTICS OF DIESEL ENGINE WITH A COTTONSEED OIL AS BIODIESEL

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Abstract- In the present work, bio-diesel is prepared from cotton seed oil and blended with diesel. The fuel properties of the synthesized bio-diesel were determined and their performance, emission and combustion characteristics were studied on a four-stroke, single cylinder, to ensure their suitability as CI engine fuel three different EGR percent's 5%,10% and 15% are used.

After conduction of experiment and obtained results I came to know that there is a steady increase in brake thermal efficiency as load increases, for standard diesel at 23 BTDC and cottonseed oil blend at different EGR percent. This is due to poor mixture formation as a result of lower volatility, higher viscosity and density of the cottonseed oil. Brake specific fuel consumption decreases from higher to lower as the Brake power increases. At the higher brake power level the brake specific fuel consumption is lower for standard diesel as well as cottonseed oil.

The NO_x emissions will reduce by increase in the EGR percentage that tends to reduce the adiabatic flame temperature and there by the combustion temperature, therefore, NO_x concentration decreases as CI engine inlet air flow is diluted with exhaust gas at a constant fuel rate. HC emissions are comparatively higher for cottonseed oil blend. As the EGR percentage increases CO emission increases and CO_2 emission decreases this is because oxygen supply is good and it reduces carbon dioxide but due to insufficient combustion carbon monoxide increases.

Key Words: Biodiesel, EGR, COME, NO_x

1. INTRODUCTION

Vitality is the life line of person leaving on the earth. Each division over the globe needs vitality, a vitality that can be renewed and more important is environmental friendly and that would not influence our environment and leaving things on the earth. Transportation is one of the areas that expend a greater amount of the vitality. These days because of the restricted assets of fossil energizes, rising unrefined petroleum prices and the increasing concerns for the

environment, there has been reestablished concentrate on the vegetable oils and creature fats as an option fuel sources. Vegetable oil fuels are not presently petroleum competitive energizes because they are more costly than petroleum fills. Nonetheless, with the recent increase in petroleum price and the uncertainties concerning petroleum accessibility, there is renewed interest in utilizing vegetable oils as a part of Diesel motors. Bio-diesel is the name for an assortment of ester-based oxygenated fuels got from common, renewable biological sources like vegetable oils. Bio-diesel works in compression ignition engines like petroleum diesel in this manner requiring no essential engine changes. Bio-diesel fuel can be produced using new or used vegetable oils and creature fats. Unlike fossil diesel, immaculate bio-diesel is biodegradable, non-toxic and essentially free of sulfur and aromatics.

The most predominantly used oil bearing crops as fuel substitutes are Sunflower, Soya bean, Cotton seed, winter Canola and Peanut. India has large waste land, which can be utilized to develop the biodiesel seed creating plants. There are number of plant species yielding oilseeds (non-consumable), which have great potential for making biodiesel and to be utilized as a fuel in conventional diesel motors with little or no modifications in basic engine outline. Biodiesel has demonstrated a lot of promise in terms of both its relatively higher combustion efficiency and lower harmful emissions. The emissions from biodiesel are diverse than petroleum-based diesel and it is important to see how they are different with respect to the levels emitted and the combustibility of the particulates. Studies conducted over the last decade have well established a direct relationship between deteriorating human health and diesel engine exhaust.

Numerous scientists have found that with biodiesel fueled motor produces higher NO_x emission contrasted with diesel. To achieve reductions in NO_x emissions, exhaust gas recirculation (EGR) can be utilized with biodiesel as a part of the diesel motors. EGR is an effective technique of decreasing NO_x emissions from the diesel engine fumes. Controlling the NO_x out flows fundamentally requires reduction of in-barrel

temperatures. However, the application of EGR results in higher fuel consumption and discharge penalties also EGR increases HC, CO, and PM emissions alongside slightly higher particular fuel consumption.

1.1 Comparison of properties of CSO, C20 and Diesel

PROPERTIES	C20	DIESEL	CSO
Carbon % (w/w)	0.192	0.2439	0.1269
Flash point °C	59	60	120
Fire point °C	68	62	153
Density g/cm ³	826	830	868
Kinematic viscosity (centistroke) at 30 °C	3.96	3.15	9.155
Specific gravity	0.821	0.83	0.876
Calorific value kJ/kg	41.46 ×10 ³	42.5 ×10 ³	37.4 ×10 ³

Properties of CSO, CS20 and diesel were carried out by different apparatus and observed that the properties of CSO is larger than the diesel and after esterification and blending with diesel the properties of C20 is getting nearer value of diesel that flash point and fire point were quite lesser than the diesel and oil is eligible for running the single cylinder diesel engine by the C20 oil.

1.2 Experimental set-up

The engine tests were conducted on a four-stroke, single-cylinder Diesel engine. The engine operated by varying injection pressure

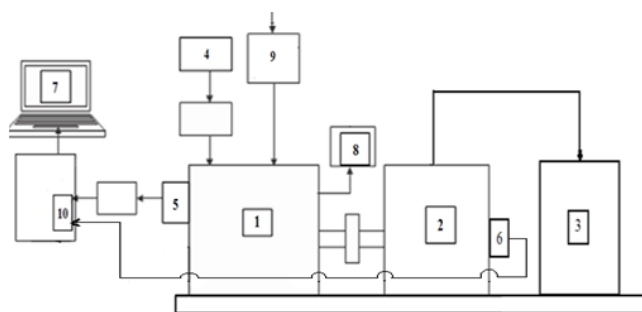


Fig.1.2. Block diagram of experimental setup

- 1. Diesel engine
- 2. Alternator
- 3. Load cell
- 4. Fuel tank
- 5. Pressure pickup
- 6. Shaft encoder
- 7. Computer
- 8. Exhaust gas analyzer
- 9. ETM meter

- 5. Pressure pickup
- 10. Data acquisition system

1.3 Engine set up

The test engine used is a single cylinder four stroke diesel engine. It produces 83.7 kW of rate power at 1500rpm with a compression ratio of 17.5:1. The engine is cooled using the water. The engine is started by hand cranking. Detailed specification of test engine is listed below in Table.

Fig shows the block diagram for engine test rig, computer was used to note down the various parameters from the experiment. The experiments are carried out at the rated speed of 1500 rpm at different load conditions. The eddy current dynamometer is used to vary the load by applying the load. Engine is first operated by the diesel oil for the heating purpose and then the different blends of fuels are used to run the engine. Rate of flow of fuels will be noted by the burette and cylinder pressure is controlled by the control panel. Exhaust gas temperature will be noted by the temperature sensors. Exhaust gas analyzers were switched on and allowed to stability before measurements.



Fig.1.3 Engine Experimental set up

1.4 Engine Specifications

1	Parameters	Specifications
2	Type	TVI (kirlosker made)
3	Software used	Engine soft.
4	Nozzle opening pressure	200-205 bar
5	Governer type	Mechanical centrifugal type
6	No of cylinders	Single cylinder
7	No of strokes	4 stroke
8	Fuel	H.S. diesel
9	Rated power	5.2 kw (7hp) at 1500 rpm
10	Cylinder diameter (bore)	87.5mm
11	Stroke length	110mm
12	Compression ratio	17.5:1
Air measurement manometer		
13	Made	MX 201
14	Type	U type
15	Range	100-0-100mm
Eddy current dynamometer		
16	Model	AG-10
17	Type	Eddy current
18	Maximum	7.5Kw at 1500-3000rpm

1.5 Exhaust gas recirculation system

Exhaust Gas Recirculation is a useful technique for reducing nitrogen oxides formation in the combustion chamber of C.I. engines. Fig 1.5 shows the arrangement of exhaust gas recirculation (EGR) system. The principle of EGR is to recirculate about 10% to 30% of the exhaust gases back into the inlet manifold where it mixes with the fresh air and this will reduce the quantity of O₂ available for combustion. This reduces the O₂ concentration and dilutes the intake charge, and reduces the peak combustion temperature inside the combustion chamber which will simultaneously reduce the NO_x formation. It should be noted that most of the NO_x emission occurs during lean mixture limits when exhaust gas recirculation is least effective. The exhaust gas which is sent into the combustion chamber has to be cooled so that the volumetric efficiency of the engine can be increased. EGR ratio is defined as the ratio of mass of recycled gases to the mass of engine intake. EGR is the most efficient and widely used system to control the formation of oxides of nitrogen inside the combustion chamber of IC engine. The exhaust gas for recirculation is taken through an orifice and passed through control valves for regulation of the quantity of recirculation.

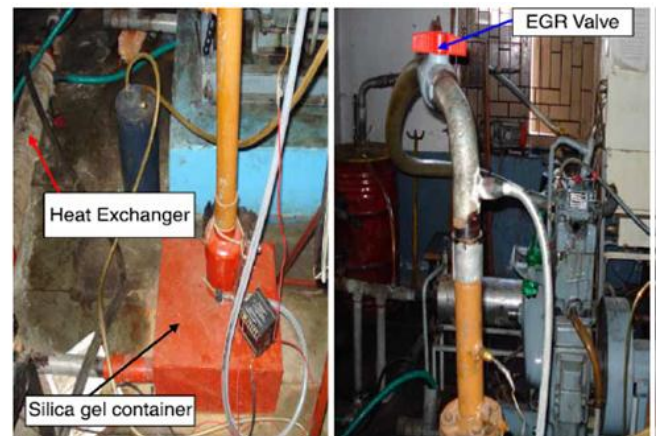


Fig 1.5 Set up of Exhaust Gas Recirculation System

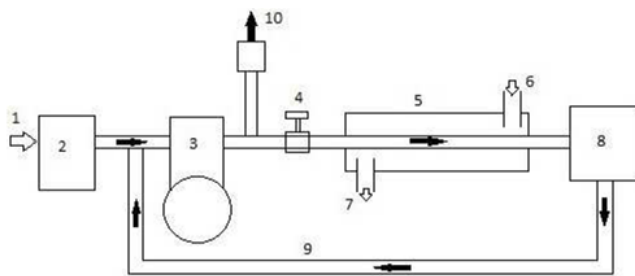


Fig.1.5 Block diagram of exhaust gas recirculation

- | | |
|-------------------|--------------------------|
| 1. Air inlet | 2. Inlet air orifice box |
| 3. Test engine | 4. EGR control valve |
| 5. Heat exchanger | 6. Water inlet |
| 7. Water outlet | 8. EGR orifice meter |
| 9. EGR pipe | 10. Exhaust gas outlet |

The EGR gas is cooled by the heat exchanger to introduce a greater amount of re-circulated gas. The silica gel container is used to absorb the carbon particles that come out through the exhaust gas. EGR valve controls the gas percentage which will be re-introduced to the engine inlet manifold.

1.6 Engine tests

Engine performance is an indication of the degree of success with which it is doing its assigned job that is the conversion of the chemical energy contained in the fuel into the useful mechanical work. The degree of success is compared on the basis of the following tests.

1. Brake thermal efficiency
2. Brake mean effective pressure
3. Brake specific fuel consumption
4. Exhaust emissions

Specific fuel consumption is widely used to compare the performance of different engines. Mean effective pressure, gives an indication of engine displacement utilization. Higher the mean effective pressure, higher will be the power developed by the engine for a given displacement. Brake thermal efficiency is the true indication of the efficiency with which the thermodynamic is converted into mechanical work. It also accounts for combustion efficiency. Exhaust emissions such as unburned hydrocarbons, carbon dioxide etc are a nuisance for the public environment. The exhaust gas percentage is varied and differentiated as 5%, 10% and 15% and the above engine tests are calculated.

2. RESULTS AND DISCUSSIONS

From the obtained results we came to know that there is a steady increase in brake thermal efficiency as load increases, for standard diesel at 23 BTDC and cottonseed oil blend at different EGR percentages. This is due to poor mixture formation as a result of lower volatility, higher viscosity and density of the cottonseed oil. Brake specific fuel consumption decreases from higher to lower as the Brake power increases. At the higher brake power level the brake specific

fuel consumption is lower for standard diesel as well as cottonseed oil.

The NO_x emissions will reduce by increase in the EGR percentage that tends to reduce the adiabatic flame temperature and there by the combustion temperature, therefore, NO_x concentration decreases as CI engine inlet air flow is diluted with exhaust gas at a constant fuel rate. HC emissions are comparatively higher for cottonseed oil blend. With the 10% EGR induction, the combustion degradation lowers the temperature. The decreased air fuel ratio associated with EGR induction and reduced combustion temperature results in incomplete fuel combustion.

Smoke emission was higher for the cottonseed oil blend than for standard diesel operation. This may be due to the heavier molecular structure, higher viscosity of the cottonseed oil. The smoke capacity for cottonseed oil was even higher with 10% EGR induction. This negative effect of EGR on smoke opacity could be due to the reduction of the engine air-fuel ratio.

2.1 Performance parameters curves

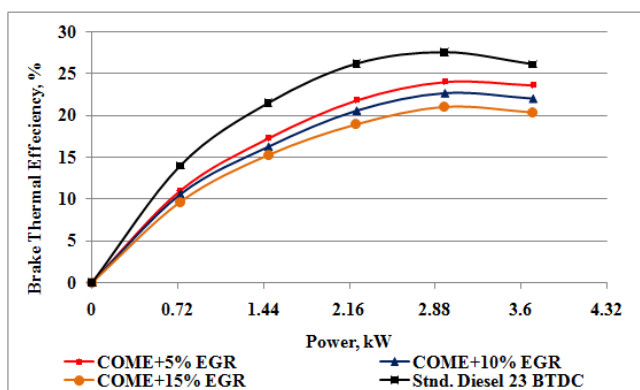


Fig 2.1 Variation of brake thermal efficiency with brake power

From the Fig 2.1 it shows that there is a steady increase in brake thermal efficiency as load increases, for standard diesel at 23 BTDC and cottonseed oil blend at different EGR percent. It was observed that the brake thermal efficiency of the engine was lower for the entire load range for cottonseed oil blend as compared to diesel operation. This is due to poor mixture formation as a result of lower volatility, higher viscosity and density of the cottonseed oil and also the implementation of EGR.

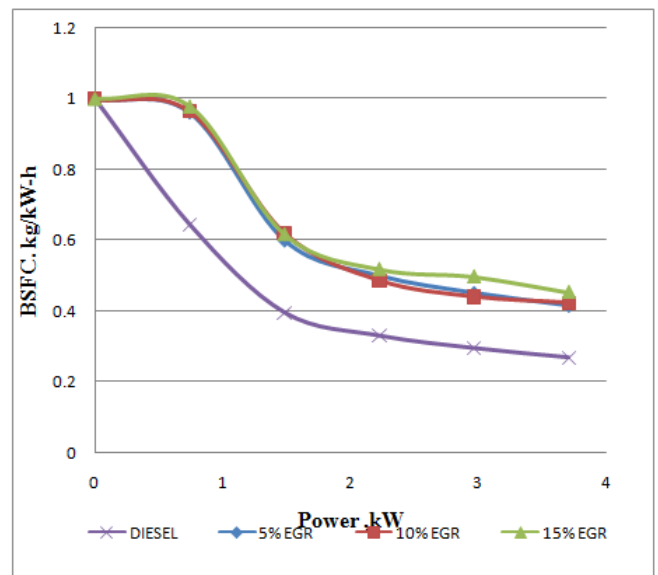


Fig 2.2 Variation of BSFC with brake power

Fig 2.2 shows the variation of brake specific fuel consumption with brake power for standard diesel and cottonseed oil blend at 5%,10%,15% EGR. The brake specific fuel consumption is defined as the ratio rate of fuel consumption to the rate of power production. Brake specific fuel consumption decreases from higher to lower as the Brake power increases. At the higher brake power level the brake specific fuel consumption is lower for standard diesel as well as cottonseed oil blend at different EGR percentages.

2.2 Emission and exhaust temperature characteristics curves

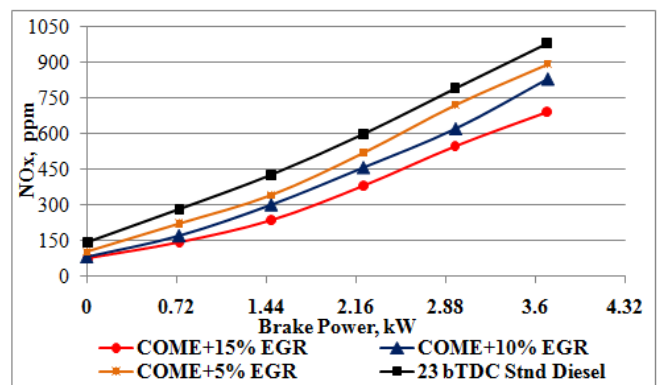


Fig 2.3 Variation of NO_x emission with brake power

Fig 2.3 shows the variation of NO_x v/s brake power, as brake power increases the NO_x emissions gradually increases. The lowest emissions of NO_x were obtained with cottonseed oil used in the EGR system engine with 15% EGR induction. The NO_x emissions will reduce by increase in the EGR percentage that tends to reduce the adiabatic flame temperature and there by the combustion temperature and hence NO_x .

emissions are reduced. Therefore, NO_x concentration decreases as CI engine inlet air flow is diluted with exhaust gas at a constant fuel rate.

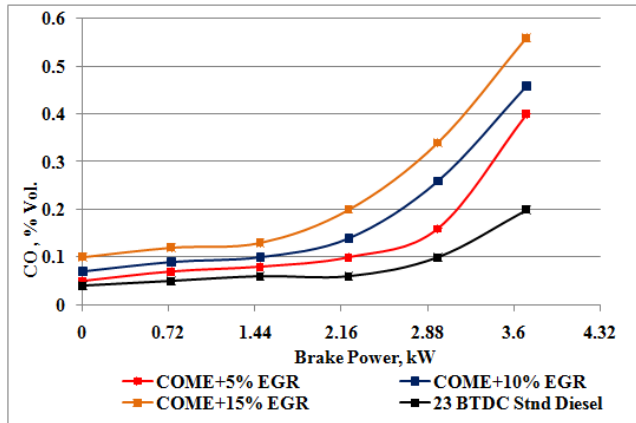


Fig 2.4 Variation of Carbon monoxide v/s Brake power

Fig 2.4 shows the effect of brake power on CO emissions. CO emissions for diesel oil are comparatively low for standard engine operation than cotton seed oil blend. For the low heat rejection operation with cottonseed oil, 10% EGR induction dilutes the combusting mixture. Therefore, insufficient oxygen is available for better combustion and the lower combustion temperature present in the engine cylinder leads to comparatively higher CO emissions. The CO at 15% EGR percentage is higher as compared to pure diesel blends without EGR.

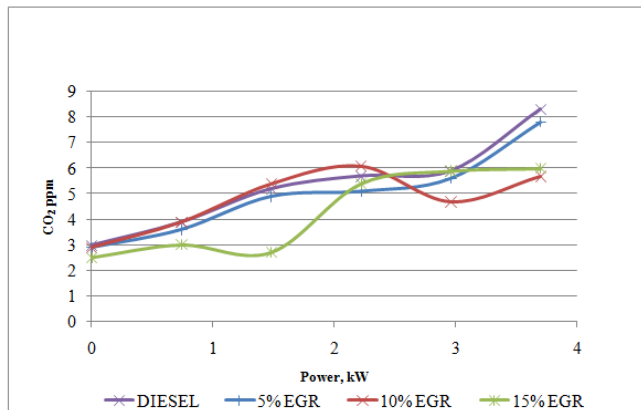


Fig 2.5 Variation of CO₂ emissions with different EGR percent

Fig 2.5 shows the effect of brake power on CO₂. CO₂ emissions for diesel oil are comparatively low for standard engine operation than cotton seed oil blends. With the 10% EGR induction, the CO₂ emission decreases this is because the oxygen supply for combustion process is increases with the increase in EGR percents. Therefore complete combustion takes place, thus reduces CO₂ emission.

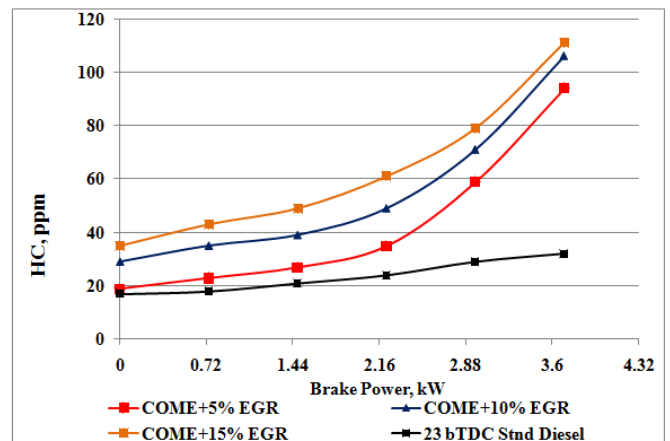


Fig 2.6 Variation of HC emission with brake power

Fig 2.6 shows the effect of brake power on HC emission and it can be observed that HC emissions are comparatively higher for cottonseed oil blend. With the 10% EGR induction, the combustion degradation lowers the temperature. The decreased air fuel ratio associated with EGR induction and reduced combustion temperature results in incomplete fuel combustion. These factors may lead to the higher HC emissions.

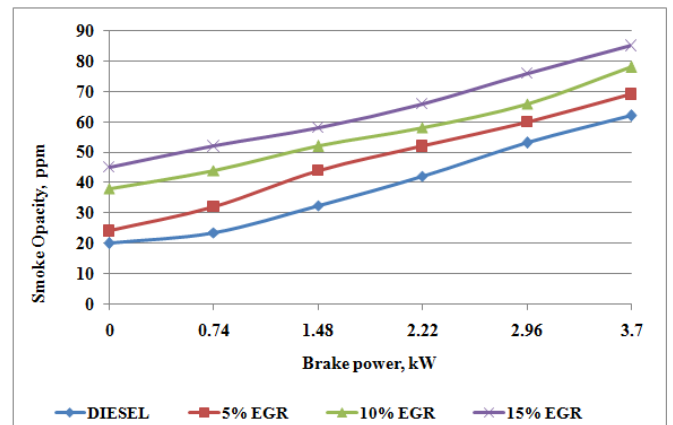


Fig 2.7 Variation of smoke capacity with brake power

Fig 2.7 shows the effect of brake power on smoke capacity. It can be observed that smoke emission was higher for the cottonseed oil blend than for standard diesel operation. This may be due to the heavier molecular structure, higher viscosity of the cottonseed oil. The smoke capacity for cottonseed oil was even higher with 10% EGR induction. This negative effect of EGR on smoke capacity could be due to the reduction of the engine air-fuel ratio.

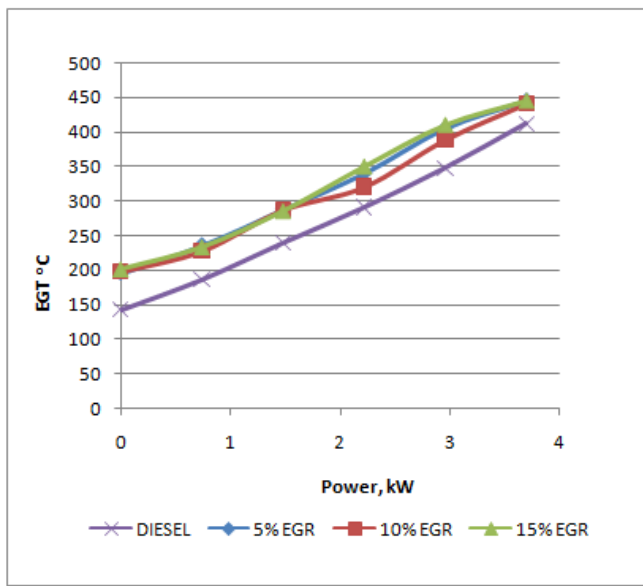


Fig 2.8 Variation of exhaust gas temperature with brake power

The exhaust gas temperature increases with increase in power. EGT increased with an increase in brake power for all fuel and engine conditions. The lowest values of EGT were obtained with standard diesel only engine operation and the highest values were observed for cottonseed oil under low heat rejection engine operation.

3. CONCLUSIONS AND SCOPE FOR FUTURE

- Biodiesel is an oxygenated fuel that undergoes improved combustion in the engine due to the presence of molecular oxygen but leads to higher NO_x emissions.
- In the experiment higher NO_x emissions is effectively controlled by 10% exhaust gas recirculation. Recycled exhaust gas lowers the oxygen concentration in the combustion chamber and increases the specific heat of intake charge which results in lower flame temperature and reduction in NO_x formation.
- Brake thermal efficiency of biodiesel is found to be comparable with diesel at all loads. The present experimental analysis on a single cylinder diesel engine with diesel and biodiesel blend at 10% EGR has proved minimized pollution and improved performance. There is an average reduction of 40% NO_x emission is obtained by 10% EGR.
- Brake specific fuel consumption decreases from higher to lower as the Brake power increases. At the higher brake power level the brake specific fuel consumption is lower for standard diesel as well as cottonseed oil blend at different EGR percentages.

- The CO at 15% EGR percentage is higher as compared to pure diesel blends without EGR.
- NO_x emissions will reduce by increase in the EGR percentage that tends to reduce the adiabatic flame temperature and there by the combustion temperature and hence NO_x emissions are reduced. Therefore, NO_x concentration decreases as CI engine inlet air flow is diluted with exhaust gas at a constant fuel rate.
- With the 10% EGR induction, the combustion degradation lowers the temperature. The decreased air-fuel ratio associated with EGR induction and reduced combustion temperature results in incomplete fuel combustion. These factors may lead to the higher HC emissions.
- The smoke capacity for cottonseed oil was even higher with 10% EGR induction. This negative effect of EGR on smoke capacity could be due to the reduction of the engine air-fuel ratio.
- The lowest values of EGT were obtained with standard diesel-only engine operation and the highest values were observed for cottonseed oil under low heat rejection engine operation.
- The NO_x and CO emission is very high than any blends. Among the different EGR, EGR with 10% has a moderate NO_x and CO emission. Thus the overall comparison shows that EGR 10% with blend 20% yields the optimum value, with less fuel consumption and higher efficiencies than diesel and it is feasible us it in the diesel engine with no modification.

3.1 SCOPE FOR FUTURE

Though many advantages of EGR, still there are some limitations which can be resolved by further modification in future. In future long term assessment of engine durability and effect on lubricating oil of bio-diesel fueled engine with EGR need to be examined. Another field of research is development of sophisticated EGR valve which could response to dynamic mode of engine operation.

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